# Beam Analysis "Analysis of Statically Determinate Beams" 

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# 1. Internal loading developed in a beam (Section Method) 

## 1. 1.1. Determine the member's support reactions

## Exemple

Determine the reactions on the beam in the Figure (a)


Solution:

(b)

Equations of equilibrium:

$$
\begin{aligned}
\xrightarrow{+} \Sigma F_{x} & =0 ; \quad A_{x}=0 \\
+\uparrow \Sigma F_{y} & =0 ; \quad A_{y}-60-60=0 \quad A_{y}=120 \mathrm{kN} \\
\dashv+\Sigma M_{A} & =0 ; \quad-60(4)-60(6)+M_{A}=0 \quad M_{A}=600 \mathrm{kN} \cdot \mathrm{~m} \\
& \xrightarrow{+} \Sigma F_{x}=0 ; \quad A_{x}=0 \\
& +\uparrow \Sigma F_{y}=0 ; \quad A_{y}-60-60=0 \quad A_{y}=120 \mathrm{kN} \\
& \downarrow+\Sigma M_{A}=0 ; \quad-60(4)-60(6)+M_{A}=0 \quad M_{A}=600 \mathrm{kN} \cdot \mathrm{~m}
\end{aligned}
$$

## 2. 1.2. Free-Body diagram

- Keep all distributed loadings, couple moments, and forces acting on the member in their exact location, then pass an imaginary section through the member, perpendicular to its axis at the point where the internal loading is to be determined.
- After the section is made, draw a free-body diagram of the segment that has the least number of loads on it. At the section indicate the unknown resultants $\mathrm{N}, \mathrm{V}$, and M acting in their positive directions.

positive sign convention


## Exemple

Determine the internal shear $(\mathrm{V})$ and moment $(\mathrm{M})$ acting at a section passing through point C in the beam shown in Figure (a)

(a)

## Solution:

## 1.Support Reactions:



## 2.Free-Body Diagram:



## 3. 1.3. Equations of Equilibrium

- Exemple

Continue with the previous example

## s.Equations Equilibrium:

$$
\begin{gathered}
+\uparrow \Sigma F_{y}=0 ; \quad 75-10 x-\left[\frac{1}{2}(20)\left(\frac{x}{9}\right) x\right]-V=0 \\
V=75-10 x-1.11 x^{2}
\end{gathered}
$$

$$
\left(+\Sigma M_{S}=0 ; \quad-75 x+(10 x)\left(\frac{x}{2}\right)+\left[\frac{1}{2}(20)\left(\frac{x}{9}\right) x\right] \frac{x}{3}+M=0\right.
$$

$$
M=75 x-5 x^{2}-0.370 x^{3}
$$

## 2. Shear Force and Moment Diagrame (S.F. D \& B.M.D)



### 1.2.1. Shear Diagram

- Establish the V and x axes and plot the values of the shear at the two ends of the beam.
- Since the slope of the shear diagram at any point is equal to the intensity of the distributed loading $w$ at the point.(Note that w is positive when it acts upward.)
- If a numerical value of the shear is to be determined at the point, one can find this value either by using the method of sections as discussed befor or by using this Eq:

$$
\begin{aligned}
\Delta V & =\int w(x) d x \\
\left.\begin{array}{rl}
\text { Change in } \\
\text { Shear }
\end{array}\right\} & =\left\{\begin{array}{l}
\text { Area under } \\
\text { Distributed Loading } \\
\text { Diagram }
\end{array}\right.
\end{aligned}
$$

, which states that the change in the shear force is equal to the area under the distributed loading diagram.

- Since $w(x)$ is integrated to obtain $V$, if $w(x)$ is a curve of degree $n$, then $\mathrm{V}(\mathrm{x})$ will be a curve of degree For example, if $\mathrm{w}(\mathrm{x})$ is uniform, $\mathrm{V}(\mathrm{x})$ will be linear.


### 2.2.2. Moment Diagram

Establish the M and x axes and plot the values of the moment at the ends of the beam.

- Since the slope of the moment diagram at any point is equal to the intensity of the shear at the point.
- At the point where the shear is zero, and therefore this may be a point of maximum or minimum moment.
- If the numerical value of the moment is to be determined at a point, one can find this value either by using the method of
sections as discussed befor or by using Eq:

$$
\begin{aligned}
\Delta M & =\int V(x) d x \\
\left.\begin{array}{c}
\text { Change in } \\
\text { Moment }
\end{array}\right\} & =\left\{\begin{array}{l}
\text { Area under } \\
\text { Shear Diagram }
\end{array}\right.
\end{aligned}
$$

which states that the change in the moment is equal to the area under the shear diagram.

- Since $V(x)$ is integrated to obtain $M$, if $V(x)$ is a curve of degree $n$, then $M(x)$ will be a curve of degree For example, if $\mathrm{V}(\mathrm{x})$ is linear, $\mathrm{M}(\mathrm{x})$ will be parabolic.


## Important useful summary

## Order of the Lines:



## Exemple

Draw the shear and moment diagrams for the beam in Figure (a) Draw the shear and moment diagrams for the beam in Figure (a)

(a)

Solution:
1.Support Reactions: the reactions have been calculated and are shown on the free-
body diagram of the beam, Figure b.

2.Shear Diagram: at points $x=0, V=+30 \mathrm{KN}$ and $\mathrm{x}=9 \mathrm{~m}, \mathrm{~V}=-60 \mathrm{KN}$ are first plotted. Note that the shear diagram (Figure c) starts with zero slope since $w=0$ at $x=0$ and ends with a slope of $w=-20 \mathrm{KN}$.

The point of zero shear can be found by using the method of sections segment of length $x$, Figure e. We require $V=0$, so that:

$$
+\uparrow \Sigma F_{y}=0 ; \quad 30-\frac{1}{2}\left[20\left(\frac{x}{9}\right)\right] x=0 \quad x=5.20 \mathrm{~m}
$$


(e)
s.Moment Diagram: for the value of shear is positive but decreasing and so the slope of the moment diagram is also positive and decreasing. At $x=5.20 \mathrm{~m}, \mathrm{dM} / \mathrm{dx}$ $=0$.

Likewise for $5.20<x<9$ the shear and so the slope of the moment diagram are negative increasing as indicated.

The maximum value of moment is at $x=5.20 \mathrm{~m}$ since $\mathrm{dM} / \mathrm{dx}=\mathrm{V}=0$ at this point,
Figure d.
From the free-body diagram in Figure $e$ we have:

$$
\begin{gathered}
\downarrow+\Sigma M_{S}=0 ; \quad-30(5.20)+\frac{1}{2}\left[20\left(\frac{5.20}{9}\right)\right](5.20)\left(\frac{5.20}{3}\right)+M=0 \\
M=104 \mathrm{kN} \cdot \mathrm{~m}
\end{gathered}
$$



